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DESIGNATED/ELECTED OFFICE (DO/EO/US)  
CONCERNING A FILING UNDER 35 U.S.C. 371

1948-4745 27123

PATENT TRADEMARK OFFICE

U.S. APPLICATION NO. (If known see 37 CFR 1.51)

09/743537

INTERNATIONAL APPLICATION  
PCT/IB00/00722

INTERNATIONAL FILING DATE  
5 May 2000 (05.05.00)

PRIORITY DATE CLAIMED  
11 May 1999 (11.05.99)

TITLE OF INVENTION

HEATING SYSTEM COMPONENTS

522 Rec'd PCT/PTO 11 JAN 2001

APPLICANT(S) FOR DO/EO/US

Noureddine KHELIFA and Horst JIRMAN

Applicant herewith submits to the United States Designated/Elected Office (DO/EO/US) the following items and other information:

1. ☒ This is **FIRST** submission of items concerning a filing under 35 U.S.C. 371.
2. ☐ This is **SECOND** or **SUBSEQUENT** submission of items concerning a filing under 35 U.S.C. 371.
3. ☒ This express request to begin national examination procedures (35 U.S.C. 371(f) at any time rather than delay examination until the expiration of the applicable time limit set in 35 U.S.C. 371 (b) and PCT Articles 22 and 39 (1).
4. ☒ A proper Demand for International Preliminary Examination was made by the 19th month from the earliest claimed priority date.
5. ☒ A copy of the International Application as filed (35 U.S.C. 371(c)(2))
  - a. ☐ is transmitted herewith (required only if not transmitted by the International Bureau).
  - b. ☒ has been transmitted by the International Bureau.
  - c. ☐ is not required, as the application was filed in the United States Receiving Office (RO/US).
6. ☒ A translation of the International application into English (35 U.S.C. 371(c)(2)) with oath.
7. ☐ Amendments to the claims of the International Application under PCT Article 19 (35 U.S.C. 371(c)(3))
  - a. ☐ are transmitted herewith (required only if not transmitted by the International Bureau).
  - b. ☐ have been transmitted by the International Bureau.
  - c. ☐ have not been made; however, the time limit for making such amendments has NOT expired.
  - d. ☐ have not been made and will not be made.
8. ☐ A translation of the amendments to the claims under PCT Article 19 (35 U.S.C. 371(c)(3)).
9. ☒ An oath or declaration of the inventor(s) (35 U.S.C. 371(c)(4)). [signed]
10. ☐ A translation of the annexes to the International Preliminary Examination Report under PCT Article 36 (35 U.S.C. 371(c)(5)).

Items 11. to 16. below concern document(s) or information included.

11. ☒ An Information Disclosure Statement under 37 CFR 1.97 and 1.98.
12. ☒ An assignment document for recording. A separate cover sheet in compliance with 37 CFR 3.28 and 3.31 is included.
13. ☒ A **FIRST** preliminary amendment.
   
☐ A **SECOND** or **SUBSEQUENT** preliminary amendment.
14. ☐ A substitute specification.
15. ☐ A change of power of attorney and/or address letter.
16. ☒ Other items or Information:
  - a. a copy of the first page of the International Application.
  - b. a copy of the International Search Report

U.S. APPLICATION NO (if known, see 37 CFR 1.51)		INTERNATIONAL APPLICATION NO		ATTORNEY'S DOCKET NO	
09/743537		PCT/IB00/00722		1948-4745	
17. <input type="checkbox"/> The following fees are submitted: <b>BASIC NATIONAL FEE</b> (37 CFR 1.492 (a) (1) - (5) ): Neither international preliminary examination fee (37 CFR 1.482) nor international search fee (37 CFR 1.445(a)(2) paid to USPTO and International Search Report not prepared by the EPO or JPO .....\$1000.00  International preliminary examination fee (37 CFR 1.482) not paid to USPTO but International Search Report prepared by the EPO or JPO ...\$860.00  International preliminary examination fee (37 CFR 1.482) not paid to USPTO but international search fee (37 CFR 1.445(a)(2) paid to USPTO .....\$710.00  International preliminary examination fee paid to USPTO (37 CFR 1.482) but all claims did not satisfy provisions of PCT Article 33 (1) - (4) .....\$690.00  9 International preliminary examination fee paid to USPTO (37 CFR 1.482) and all claims satisfied provisions of PCT Article 33(1) - (4) .....\$100.00  ENTER APPROPRIATE BASIC FEE AMOUNT =				CALCULATIONS PTO USE ONLY	
Surcharge of \$130 for furnishing the oath or declaration later than <input type="checkbox"/> 20 <input type="checkbox"/> 30 months from the earliest claimed priority date (37 CFR 1.492(e)).				\$ 0	
CLAIMS	NUMBER FILED	NUMBER EXTRA	RATE	\$860.00	
Total claims	- 20 =		X \$18.00	\$ 0	
Independent claims	1 - 3 =	0	X \$78.00	\$ 0	
MULTIPLE DEPENDENT CLAIM(S) (if applicable)			+ \$260.00	\$ 0	
<b>TOTAL OF ABOVE CALCULATIONS =</b>				\$860.00	
Reduction of 1/2 for filing by small entity, if applicable. A Small Entity Statement must also be filed (Note 37 CFR 1.9, 1.27, 1.28).				\$ 0	
<b>SUBTOTAL =</b>				\$860.00	
Processing fee of \$130.00 for furnishing the English translation later than <input type="checkbox"/> 20 <input type="checkbox"/> 30 months from the earliest claimed priority date (37 CFR 1.492(f)).				\$ 0	
<b>TOTAL NATIONAL FEE =</b>				\$860.00	
Fee for recording the enclosed assignment (37 CFR 1.21(h)). The assignment must be accompanied by an appropriate cover sheet (37 CFR 3.28, 3.31). \$40.00 per property				\$ 40.00	
<b>TOTAL FEES ENCLOSED</b>				\$900.00	
				Amount to be refunded:	\$
				charged	\$
a. <input checked="" type="checkbox"/> A check in the amount of \$ to cover the above fees is enclosed.					
b. <input type="checkbox"/> Please charge my Deposit Account No. in the amount of \$ to cover the above fees.					
c. <input checked="" type="checkbox"/> The Commissioner is hereby authorized to charge any additional fees which may be required, or credit any overpayment to Deposit Account No. 13-4500. Order No. 1948-4745. A duplicate copy of this sheet is enclosed.					
<b>NOTE:</b> Where an appropriate time limit under 37 CFR 1.494 or 1.495 has not been met, a petition to revive (37 CFR 1.137(a) or (b)) must be filed and granted to restore the application to pending status.					
Date: 1/11/01					
SEND ALL CORRESPONDENCE TO.					
Morgan & Finnegan, LLP 345 Park Avenue New York, NY 10154-0053					
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SIGNATURE Joseph A. Calvaruso NAME 28,287 REGISTRATION NO.					

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11 JAN 2001

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## Application Information

Title Line One :: HEATING SYSTEM COMPONENTS  
Total Drawing Sheets :: 8  
Formal Drawings :: Y  
Application Type :: Utility  
Docket Number :: 1948-4745

## Representative Information

Representative Customer Number :: 27,143  
Registration Number One :: 28,287

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Prior Foreign Applications

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Filing Date :: 11 May 1999  
Country :: Germany  
Priority Claimed :: Y

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IN THE UNITED STATES PATENT AND TRADEMARK OFFICE (DO/EO/US)

5 Applicants : Nouredine KHELIFA and  
Horst JIRMANN

International  
Application No. : PCT/IB00/00722

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U.S. Serial No. : To be assigned

15 U.S. Filing Date : 11 January 2001 (11.01.01)

Group Art Unit : To be assigned

Examiner : To be assigned

20 For : HEATING SYSTEM COMPONENTS

PRELIMINARY AMENDMENT

25 Commissioner For Patents  
Box PCT  
Washington, D.C. 20231

30 Attention: DO/EO/US

S I R:

Preliminary to a substantive examination on the  
merits, please amend the above identified application as

35 follows:

IN THE ABSTRACT

An abstract is submitted on a separate sheet of  
paper, to be appended as page 17 of the patent application as  
submitted herewith.

40 IN THE CLAIMS

Amend claims 1 through 8 as follows:

1. (amended) A [H]heating and air conditioning  
installation[,particularly,] for a vehicle having a first  
fluid circuit [(10)] comprising a first heat exchanger [(12)],

a second fluid circuit [(20)] comprising a second heat exchanger[(22)], and having a third fluid circuit [(30)] comprising a third heat exchanger [(32)], [characterized in that] wherein the second and/or third heat exchanger [(22, 32)] is connected downstream of the first heat exchanger [(12)] in terms of air-flow technology.

2. (amended) A [H]heating and air conditioning installation according to Claim 1, [characterized in that] wherein the second or the third heat exchanger [(22, 32)] is connected downstream of the first heat exchanger [(12)] in terms of air-flow technology, the other heat exchanger [(22, 32)] being connected or connectable upstream of the first heat exchanger [(12)].

3. (amended) A [H]heating and air conditioning installation according to Claim 1 [or 2, characterized in that], wherein the downstream heat exchanger(s) [22, 32]] forms/form a structural unit with the first heat exchanger [(12)], the respective heat exchangers [(12, 22, 32) in particular] being thermally isolated from one another.

4. (amended) A [H]heating and air conditioning installation according to [one of the preceding claims characterized in that] Claim 1, wherein the second and the third fluid circuits [(20, 30)] are coupled to one another,

one of said second and third fluid circuits [(20, 30) in particular] forming a bypass to the other fluid circuits [(20, 30)].

5           5. (amended) A [H]heating and air conditioning installation according to Claim 4 [characterized in that], wherein a fluid-flow and/or state control member [means (31; 34; 35. 36; 35, 37, 38)] are provided in [the] a transition region between the second and third fluid circuit [(20, 30)].

10           6. (amended) A [H]heating and air conditioning installation according to Claim 5 [characterized in that], wherein at least one fluid-flow and/or state control member [means (31; 34; 35, 36; 35, 37, 38) fulfils the function of a  
15 throttle, in particular designed] acts as a two- or multi-stage throttle [(38)].

20           7. (amended) A [H]heating and air conditioning installation according to [one of the preceding claims characterized in that] Claim 1, wherein an air-flow control [means] member is connected upstream of at least one of said first, second and third heat exchangers [(12, 22, 32), in particular] for diverting or delivering air with respect to  
25 [the]said at least one heat exchanger [(12, 22, 32) in question].

8. (amended) A [H]heating and air conditioning  
installation according to [one of the preceding claims  
characterized in that] Claim 1, wherein a phase change of the  
fluid takes place in at least one of said first, second and  
5 third heat exchanger [(12, 22, 32)].

Add new claim 9 as follows:

9. A heating and air conditioning installation according  
10 to Claim 2, wherein the downstream heat exchanger(s)  
forms/form a structural unit with the first heat exchanger,  
the respective heat exchangers being thermally isolated from  
one another.

15 **REMARKS**

An Abstract is submitted on a separate sheet of  
paper to be appended to page 17 of this patent application.

New claims have been presented to better describe  
the invention and eliminate multiple dependent claims. The  
20 claims have been amended to delete character references.

No new matter was added to the application as a  
result of this amendment.



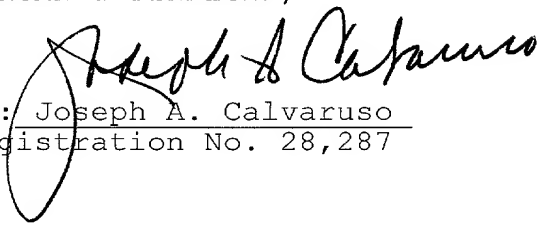
The Commissioner is hereby authorized to charge any additional fees which may be required for this amendment, or credit any overpayment to Deposit Account No. 13-4500, Order No. 1948-4745.

5

Respectfully submitted,

MORGAN & FINNEGAN, L.L.P.

Dated: January 11, 2001

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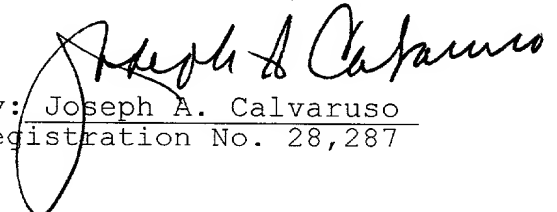
The Commissioner is hereby authorized to charge any additional fees which may be required for this amendment, or credit any overpayment to Deposit Account No. 13-4500, Order No. 1948-4745.

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09/743537

JC07 Rec'd PCT/PTO 11 JAN 2001

I, Eur Ing James Francis Bradley BSc CEng MIEE MIL,  
translator to Engineering Translations, of  
"Hollyhocks", Cherry Tree Road, Farnham Royal,  
Buckinghamshire, England, hereby declare that I am  
conversant with the German and English languages and am  
a competent translator thereof. I declare further that  
to the best of my knowledge and belief the following is  
a true and correct translation of the accompanying  
International Patent Application No. PCT/IB00/00722 in  
the German language.

Signed this 18<sup>th</sup> day of December 2000



J. F. BRADLEY

For and on behalf of Engineering Translations

Heating-System Components

09/743537

JC07 Rec'd PCT/PTO 11 JAN 2001

The present invention relates in general to a heating and air conditioning installation, and in particular to a heating and air conditioning installation for a vehicle.

A conventional heating and air conditioning installation for a vehicle is usually formed by an engine-side coolant circuit and a refrigerant circuit. For a clearer understanding of the invention described below, the term "first fluid circuit" should be understood as meaning the engine-side coolant circuit. In normal heating and air conditioning installations, as well as in the case of the present invention, various heat exchangers, condensers and evaporators are used. In the description of the invention below and in the appended claims, the term "heat exchanger" should be understood as meaning heat exchangers, evaporators and condensers involved in the air conditioning; in other words, the term "heat exchanger" is intended to cover only the components directly involved in the air conditioning, and not units which are used to deliver or take heat to or from the surrounding air, such as a heat sink which is used to dissipate the waste engine heat, for example.

Taking into account the definitions above, a normal heating and air conditioning installation is made up of a first fluid circuit comprising a first heat exchanger, and a second fluid circuit comprising a second heat exchanger, the heat exchanger of the first fluid circuit usually being used for heating purposes, whereas the heat exchanger of the second fluid circuit, usually designed as an evaporator, is used for cooling.

In order to improve the heating performance of such heating and air conditioning installations, a va-

riety of attempts have been made to raise the heating performance of the first heat exchanger by providing additional equipment, such as e.g. PTC heating registers, fuel cells, etc. These approaches are, however, 5 relatively complex to implement and entail relatively high overall fitting costs for the heating and air conditioning installation.

US 5,291,941 has therefore proposed a heating and air conditioning installation in which the second 10 fluid circuit can be used for both cooling and heating purposes. As a further development, this document also describes a heating and air conditioning installation which comprises three fluid circuits, namely a first fluid circuit which provides heat from the internal 15 combustion engine for heating purposes via a first heat exchanger, a second fluid circuit which is configured as a conventional refrigerant circuit for cooling, and a third fluid circuit, which is designed as a thermodynamic heating circuit. This preferred embodiment corresponds to the precharacterizing clause of Claim 1. 20

Although this known heating and air conditioning installation enables optimization of the second and third heat exchangers for cooling and heating purposes, respectively, this embodiment has nevertheless been 25 found to be inadequate with respect to currently required response times and control facilities.

The object of the present invention is therefore to refine an air conditioning installation of the generic type, in particular for a vehicle having a first 30 fluid circuit comprising a first heat exchanger, a second fluid circuit comprising a second heat exchanger, and having a third fluid circuit comprising a third heat exchanger, in such a way as to provide shorter response times and an improved control facility.

According to the invention, this object is achieved in that the second and/or third heat exchanger is connected downstream of the first heat exchanger in terms of air-flow technology. In contrast to the prior art, it is hence possible to connect at least one heat exchanger downstream of the heat exchanger of the engine-side coolant circuit in terms of air-flow technology, so that e.g. when the engine is cold in the phase of starting the vehicle, the heating power of the assigned thermodynamic heating circuit is made available virtually immediately, since no part of the heat generated when heating up the first heat exchanger is lost. In heating operation, in particular, the downstream connection of a further heat exchanger of a thermodynamic heating circuit offers an optimized control facility, since the final adjustment of the air to be delivered into the passenger compartment can be carried out by means of this heat exchanger.

In an advantageous refinement of the heating and air conditioning installation, three heat exchangers are arranged in series in terms of air-flow technology, so that the first heat exchanger of the engine-side fluid circuit is arranged between the second and third heat exchangers. For example, air blown by a fan could thus first pass through an evaporator intended for cooling purposes, then through the heat exchanger of the coolant circuit and then through the heat exchanger of a thermodynamic fluid circuit designed for heating purposes. The fact that the heat exchanger used for cooling purposes is connected upstream of the heat exchangers used for heating purposes means that the moisture formed by the heat exchanger used for cooling purposes no longer constitutes a problem, even in heating mode. This moisture formation was particularly problematic in the prior art, and frequently caused the win-

dows to fog up in the motor vehicle. However, since the air passing through the heat exchanger used for cooling purposes is at a relatively low temperature, this air takes up only a little moisture. The result of this is that, after the heating which follows by means of the first heat exchanger of the engine-side fluid circuit and the subsequent final heating by means of the downstream heat exchanger of a thermodynamic fluid circuit, the total humidity of the emerging air is negligible, which means that the windows no longer fog up. By providing an upstream heat exchanger used for cooling purposes and a downstream heat exchanger used for heating purposes, with respect to the first heat exchanger of the engine-side fluid circuit, it is furthermore possible to optimize the configurations of the second and third heat exchangers. This makes it possible to avoid the noise emissions which result in the prior art when only one heat exchanger in addition to the engine-side heat exchanger is used for both cooling and heating purposes.

Advantageously, the downstream heat exchanger or heat exchangers are formed as a structural unit with the first heat exchanger, the respective heat exchangers being in particular thermally isolated from one another. This arrangement makes it possible to combine the heat exchangers used for heating purposes in a compact way while maintaining thermal isolation.

In order to reduce the number of components necessary, for instance fluid delivery means and the like, it is preferable if the second and the third fluid circuits are coupled to one another, one fluid circuit, in particular, representing a bypass with respect to the other fluid circuit. In this way, the two thermodynamic fluid circuits can be combined into an overall circuit which can be used selectively, by means of suitable



control elements, for heating and/or for cooling. An advantage of the bypass-type configuration is that no reversal is any longer necessary when changing between cooling and heating operations.

5           Advantageously, fluid-flow and/or state control means are provided in the transition region between the second and third fluid circuits. These means may be used, on the one hand, to switch the respective mode between heating and cooling and, on the other, for controlling the power level of the corresponding operating mode. The fluid-flow control means and the fluid-state control means may be designed as combined equipment, for instance in the form of a three-way valve, one outlet being designed as a throttle, or alternatively as  
10           separate equipment, for instance control valves in conjunction with separate throttle points, e.g. in the form of a throttle valve or a cross-sectional constriction in the delivery line.  
15

          In a particularly advantageous embodiment, at least one fluid-flow and/or state control means is arranged in such a way that it can affect both the fluid flow and the fluid state, e.g. designed as a throttle, it being particularly advantageous if the throttle function is achieved by forming a two- or multi-stage  
20           throttle. In the multi-stage version a large resistance might for instance be selected when the thermodynamic fluid circuit is being heated, and switched over to a state with low resistance when the heat required to achieve the desired heating performance is sufficient.  
25

30           In order to improve yet further the control facility when switching between cooling and heating modes, it may be advantageous to provide separate control means for the individual fluid circuits, for instance in the form of simple shut-off valves or via a

non-return valve, which may optionally be provided with a temperature controller.

In order to improve further the response of the heating and air conditioning installation according to the invention, an air-flow control means may be connected upstream of at least one heat exchanger, in particular for diverting or delivering air with respect to the heat exchanger in question. In other words, in this configuration there are means for providing an air by-pass for one or more heat exchangers. In cooling mode, it might for instance be advantageous to deliver the air exclusively through the evaporator, i.e. the heat exchanger used for cooling purposes, without it flowing through the heat exchangers used for heating purposes, which might possibly still have a certain amount of residual heat stored. When subsequently switching over from cooling mode to heating mode, the flow through the heat exchanger used for cooling purposes could be stopped, so that the air flows exclusively through the heat exchanger used for heating purposes. Accordingly, the available heating power is not compromised by the residual cold in the heat exchangers used for cooling purposes. The response is hence improved significantly, and in particular the heat exchanger used for heating purposes may be able to retain residual heat since it has not been cooled in the cooling mode.

Lastly, it is preferable if a phase change of the fluid takes place in at least one heat exchanger. In other words, at least one heat exchanger should be designed as an evaporator or a condenser, so that the fluid circuit of this heat exchanger can be regarded as a thermodynamic heating or cooling circuit.

Other advantages and features of the present invention will be found in the following detailed description of a few currently preferred, purely illus-

trative embodiments, which is given with reference to the appended drawings, in which:

Fig. 1 shows a first preferred embodiment of the heating and air conditioning installation according to the invention, in which three discrete fluid circuits are provided.

Fig. 2 shows a second preferred embodiment of the heating and air conditioning installation according to the invention, two fluid circuits being combined or coupled with one another.

Fig. 3 shows a further preferred embodiment, similar to the embodiment shown in Fig. 2, the heat exchangers involved in the air conditioning being arranged in series.

Fig. 4 shows two heat exchangers which are combined with one another as a structural unit, and which form part of a preferred embodiment of the invention.

Fig. 5 shows a detailed view of heat exchangers arranged in series, with assigned air-flow control means.

Figs. 6a to e show various possible fluid-flow and/or state control means for coupled fluid circuits.

Fig. 7 diagrammatically shows the thermodynamic fluid circuits of a particularly preferred embodiment of the heating and air conditioning installation according to the invention.

Fig. 1 diagrammatically represents a first preferred embodiment of the heating and air conditioning installation according to the invention. The heating and air conditioning installation comprises a first fluid circuit 10 comprising a first heat exchanger 12. This first fluid circuit 10 represents the engine-side cooling circuit which, for example, may be used to cool an internal combustion engine 1. The components of the first fluid circuit 10 are conventional and it com-

prises, besides the heat exchanger 12 used for heating purposes, fluid delivery means 5, a bypass line 2 controlled by means of a thermostat 3, and a heat sink 4 which interacts with the surrounding air. A fan or  
5 blower 14 is provided for application to the heat exchanger 12 in terms of air-flow technology.

In some driving situations, in particular after a cold start, the heating power provided by the heat exchanger 12 is not sufficient to bring the vehicle interior to a desired temperature. In the embodiment  
10 shown, a further heat exchanger 32, which is identified as the third heat exchanger 32 in the description below, is therefore connected downstream of the heat exchanger 12 in terms of air-flow technology. The third  
15 heat exchanger 32 is supplied via a thermodynamic heating circuit 30 which, in the conventional way, comprises fluid-application means 33 and fluid-expansion means 36. The downstream connection of the third heat exchanger 32 accordingly makes it possible to optimize  
20 the heating performance which is set, in terms of control technology; for example, after a cold start it is possible to achieve improved response if the third heat exchanger 32 can supply heat even when the heat exchanger 12 of the first fluid circuit is still cold.

25 In order to make it possible, when required, to obtain air conditioning or possibly also dehumidification, a heat exchanger 22 used for cooling purposes is arranged upstream of the first heat exchanger 12 in terms of air-flow technology. This heat exchanger 22  
30 used for cooling purposes will be defined below as the second heat exchanger 22, and is usually designed in the conventional way as an evaporator. The second heat exchanger 22 is supplied through a second thermodynamic fluid circuit which, in the usual way in these instances,  
35 comprises fluid-delivery and/or application

means 25 and fluid-expansion and/or throttle means 26. The second fluid circuit 20 may optionally also comprise a heat sink 24, which is used to supply the surrounding air with heat which is not required for controlling the temperature of the passenger compartment in pure cooling operation. This heat sink 24 is designed in the conventional way as a condenser.

The series arrangement of heat exchangers, in the sequence of second heat exchanger 22, first heat exchanger 12 and third heat exchanger 32, makes it possible to achieve optimum control of the temperature of the air to be fed into the passenger compartment. In the embodiment represented here, the temperature of the air is controlled exclusively by means of fluid-technology control of the three fluid circuits, although additional air-flow control means may also be provided, as discussed further on in the present description.

It can be stated that the downstream connection of the third heat exchanger allows an optimized configuration of the heat exchanger for the fluid for heating purposes, with the additional possibility of re-warming and hence optimized temperature control of the air entering the passenger compartment. In this way, it is possible to eliminate noise emission problems which result in the prior art if only one heat exchanger is used for cooling and heating purposes. The upstream arrangement of the second heat exchanger with respect to the first heat exchanger 12 makes it possible to cool the air entering the passenger compartment in a suitable way, switching from cooling mode to heating mode being readily possible without the moisture present in the second heat exchanger 22 having the potential to constitute a problem. The upstream connection of the second heat exchanger 22 in terms of air-

flow technology does provide an optimized solution in terms of compactness, but it is also possible to provide the heat exchanger 22 used for cooling in a separate channel or in parallel with the first heat exchanger 12. Such an embodiment is represented in Fig. 2.

In the embodiment represented in Fig. 2, the second fluid circuit 20 and the third fluid circuit 30 are represented as a coupled fluid circuit, so that only one fluid-delivery and/or application means 25 is needed for the two circuits. As represented, the third fluid circuit with the heat exchanger 32 used for subsequent temperature control forms a bypass with respect to the second fluid circuit 20. In order to switch between heating and cooling modes, for example, two three-way valves 35 are simply operated for application to the second fluid circuit 20 or the third fluid circuit 30. It should be mentioned that, if a certain degree of dehumidification is desired, mixed operation may also be carried out. In other words, the control valves 35 are set in such a way that both the heat exchanger 32 and the heat exchanger 22 are supplied with fluid, on one hand in order to provide cooling in order to dry the air and, on the other, in order to provide heating which is used to raise the air entering the passenger compartment to a desired temperature immediately before it emerges.

Fig. 3 represents a third preferred embodiment, the structure of which corresponds essentially to the embodiment represented in Fig. 2. As in the embodiment represented in Fig. 2, the two thermodynamic fluid circuits 20 and 30 are coupled to one another, although in this embodiment the heat exchangers 22, 12, 32 are arranged in series, specifically in a similar way to the embodiment represented in Fig. 1. The advantages of the

embodiments shown in Figs. 1 and 2 can accordingly be achieved in combination.

Fig. 4 represents the first heat exchanger 12 and the third heat exchanger 32, i.e. the two heat ex-  
5 changers used for heating purposes, as a combination unit, the design of the unit being such that thermal decoupling exists between the two heat exchangers. This arrangement of the heat exchangers in the form of a tandem radiator allows compact fitting. As is repre-  
10 sented in Fig. 4, further thermal decoupling can take place if the feed lines for coolant or refrigerant are offset relative to one another.

The form of the tandem radiator may, however, also be configured in another suitable fashion, for ex-  
15 ample as a hollow body which, in segmented fashion, in sections or as mentioned above in thermally separated and parallel-running fashion, is applied in co-current or counter-current flow by the corresponding fluid circuits.

Fig. 5 represents the series arrangement of the  
20 heat exchangers 22, 12, 32 in conjunction with air-flow control means, which make it possible to feed air through the respective heat exchangers or divert air past them. This arrangement makes it possible to rein-  
25 force or replace the fluid-side control; in any case the response is improved since residual heat and residual cold can respectively remain in the relevant heat exchangers when not being used. In terms of flow technology, one channel 40 contains, downstream of the fan  
30 14, the heat exchanger 22 used for cooling purposes and the heat exchangers 12 and 32 used for heating purposes, which are provided as a combination unit. Two control flaps 42 and 44 which, as represented, may enable or prevent application to the heat exchanger 22,  
35 are arranged upstream of the heat exchanger 22 used for

cooling purposes. Correspondingly, the air is fed either through the heat exchanger 22 or past the latter in bypass fashion. Correspondingly, an air-flow control means with two flaps 46 and 48 is arranged upstream of the combination unit consisting of the two heat exchangers 12, 32 used for heating purposes, so that the air can be caused to pass through these heat exchangers or be fed past them. Since the application to the individual heat exchangers can be chosen selectively according to the operating mode, the responsiveness of the overall circuit may be significantly improved, especially since, during cooling, the residual heat contained in the heat exchangers 12, 32 used for heating purposes does not compromise the cooling power, whereas during heating, the heat exchanger 22 used for cooling need not be heated up initially. When switching between cooling and heating modes, the remaining residual heat or residual cold can hence advantageously be used immediately.

Figs. 6a to e represent various possible ways in which the thermodynamic second fluid circuit 20 used for cooling purposes and the third fluid circuit 30 used for heating purposes, which [sic] may be coupled to one another, various possible ways of expanding the fluid in the third thermodynamic heating circuit 30 being in particular represented.

In Fig. 6a, a three-way valve 35, which can selectively apply to one of the fluid circuits 20, 30 or both fluid circuits 20, 30 in respective proportions, is connected downstream of the fluid-delivery means, or compressor, 25 in terms of fluid-flow technology. For the third fluid circuit 30 used for heating purposes, a throttle point 36 in the form of an expansion valve is arranged downstream of the three-way valve 35. The third heat exchanger 32, which is used as an after-



heater the for the engine-side fluid-circuit heat exchanger, is located further downstream. A non-return valve is furthermore provided at the interface with the second fluid circuit 20 used for cooling purposes.

5           In the embodiment shown in Fig. 6b, the throttle point 36 and the three-way valve 35 are combined to form one unit 34, in which the outlet directed toward the third fluid circuit 30 is used as a throttle.

10           In the embodiments represented in Figs. 6c and 6d, the throttle function respectively is achieved by two valves 36 and by hoses or lines with reduced diameters 37, it being possible for corresponding throttling of the fluid used for heating purposes to take place on both sides of the heat exchanger 32.

15           In the embodiment shown in Fig. 6e, two three-way valves, the respective inlet or outlet of which are designed as throttle points toward the third fluid circuit 30, are provided one on each side of the fluid-delivery or application means 25.

20           Finally, Fig. 7 shows another possible combination between the second and third thermodynamic fluid circuits 20, 30, in a representation corresponding essentially to that in Fig. 2. In order to increase the control facility, in the embodiment represented here a pair of control valves 35 for the respective fluid circuits is connected downstream of the compressor 25. In the embodiment represented here, a two-stage throttle, which has slight resistance when there is enough heat present in the thermodynamic heating circuit 30, is connected downstream of the heat exchanger 32 used for heating purposes. In the heating phase, for example after a cold start of the vehicle, the resistance can be adjusted to measure, so that rapid heating can be carried out by means of the throttle 38 with high resistance. Consequently, the heat exchanger 32 can provide

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heat for the passenger-compartment heating system just a short time after starting, so that a satisfactory heating function can be fulfilled almost immediately after the start.

5           In conclusion, it may be stated that the solution according to the invention provides a heating and air conditioning installation which enables rapid response to a very wide variety of settings, substantially independently of external conditions and states  
10 in the system. In contrast to other approaches, such as when PTC heating registers or fuel cells are used, only known thermodynamic components are required in the proposed solution; it is advantageously possible to reduce the number of parts required further if the thermodynamic  
15 fluid circuits are coupled or connected to one another. In this way, a facility for optimized control of the overall system is provided, without excessive space being required.

          Although the present invention has been described  
20 scribed above completely and with reference to the appended drawings, the person skilled in the art should realize that a very wide variety of changes and modifications are possible in the scope of the claims. In particular, it should be pointed out that the individual  
25 ual features of one embodiment can be combined at will with other features of other embodiments.

Claims

1. Heating and air conditioning installation, particularly, for a vehicle having a first fluid circuit  
5 (10) comprising a first heat exchanger (12), a second fluid circuit (20) comprising a second heat exchanger (22), and having a third fluid circuit (30) comprising a third heat exchanger (32), characterized in that the second and/or third heat exchanger (22, 32) is con-  
10 nected downstream of the first heat exchanger (12) in terms of air-flow technology.
2. Heating and air conditioning installation according to Claim 1, characterized in that the second or the third heat exchanger (22, 32) is connected down-  
15 stream of the first heat exchanger (12) in terms of air-flow technology, the other heat exchanger (22, 32) being connected or connectable upstream of the first heat exchanger (12).
3. Heating and air conditioning installation according to Claim 1 or 2, characterized in that the  
20 downstream heat exchanger(s) (22, 32) forms/form a structural unit with the first heat exchanger (12), the respective heat exchangers (12, 22, 32) in particular being thermally isolated from one another.
- 25 4. Heating and air conditioning installation according to one of the preceding claims, characterized in that the second and the third fluid circuits (20, 30) are coupled to one another, one fluid circuit (20, 30) in particular forming a bypass with respect to the  
30 other fluid circuits (20, 30).
5. Heating and air conditioning installation according to Claim 4, characterized in that fluid-flow and/or state control means (31; 34; 35, 36; 35, 37, 38) are provided in the transition region between the sec-  
35 ond and third fluid circuit (20, 30).

6. Heating and air conditioning installation according to Claim 5, characterized in that at least one fluid-flow and/or state control means (31; 34; 35, 36; 35, 37, 38) fulfils the function of a throttle, in particular designed as a two- or multi-stage throttle (38).

7. Heating and air conditioning installation according to one of the preceding claims, characterized in that an air-flow control means is connected upstream of at least one heat exchanger (12, 22, 32), in particular for diverting or delivering air with respect to the heat exchanger (12, 22, 32) in question.

8. Heating and air conditioning installation according to one of the preceding claims, characterized in that a phase change of the fluid takes place in at least one heat exchanger (12, 22, 32).

Fig. 1

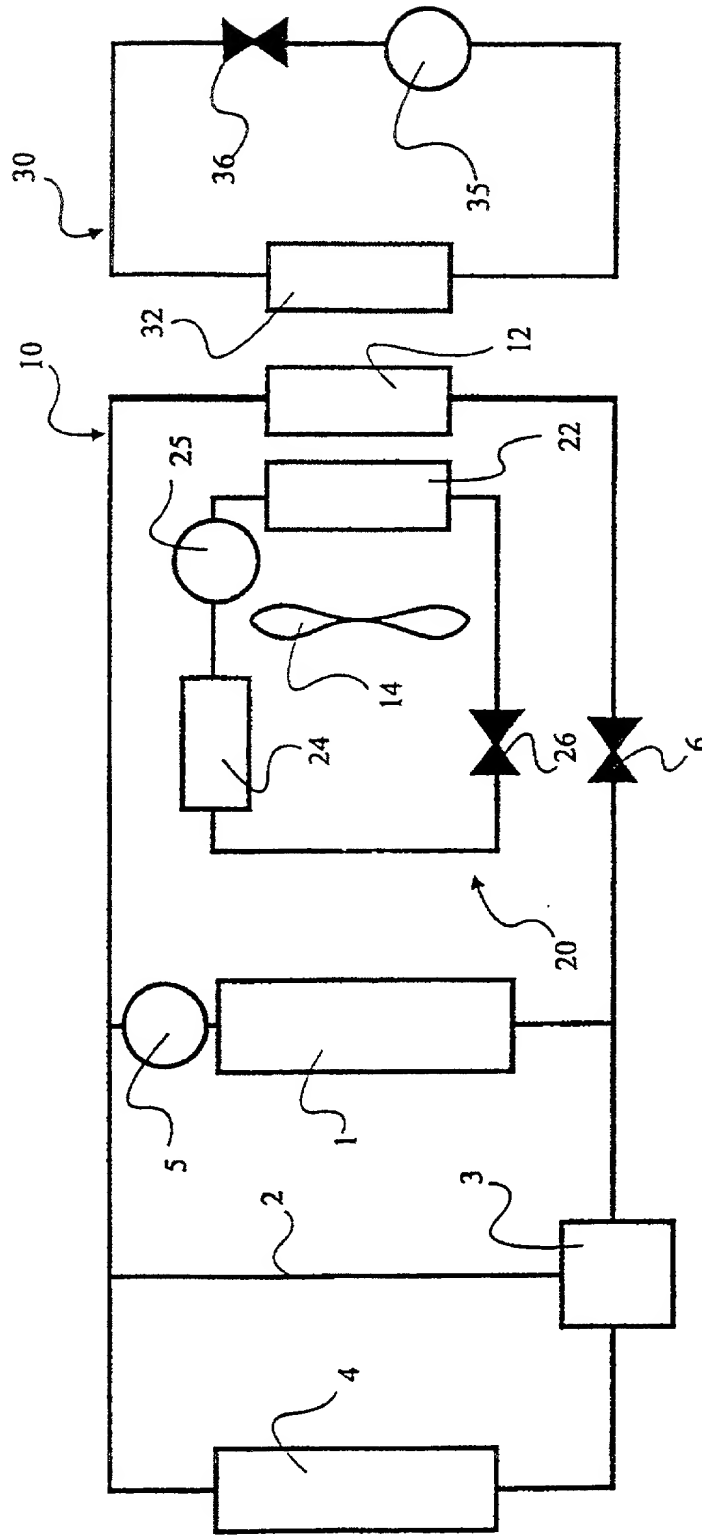
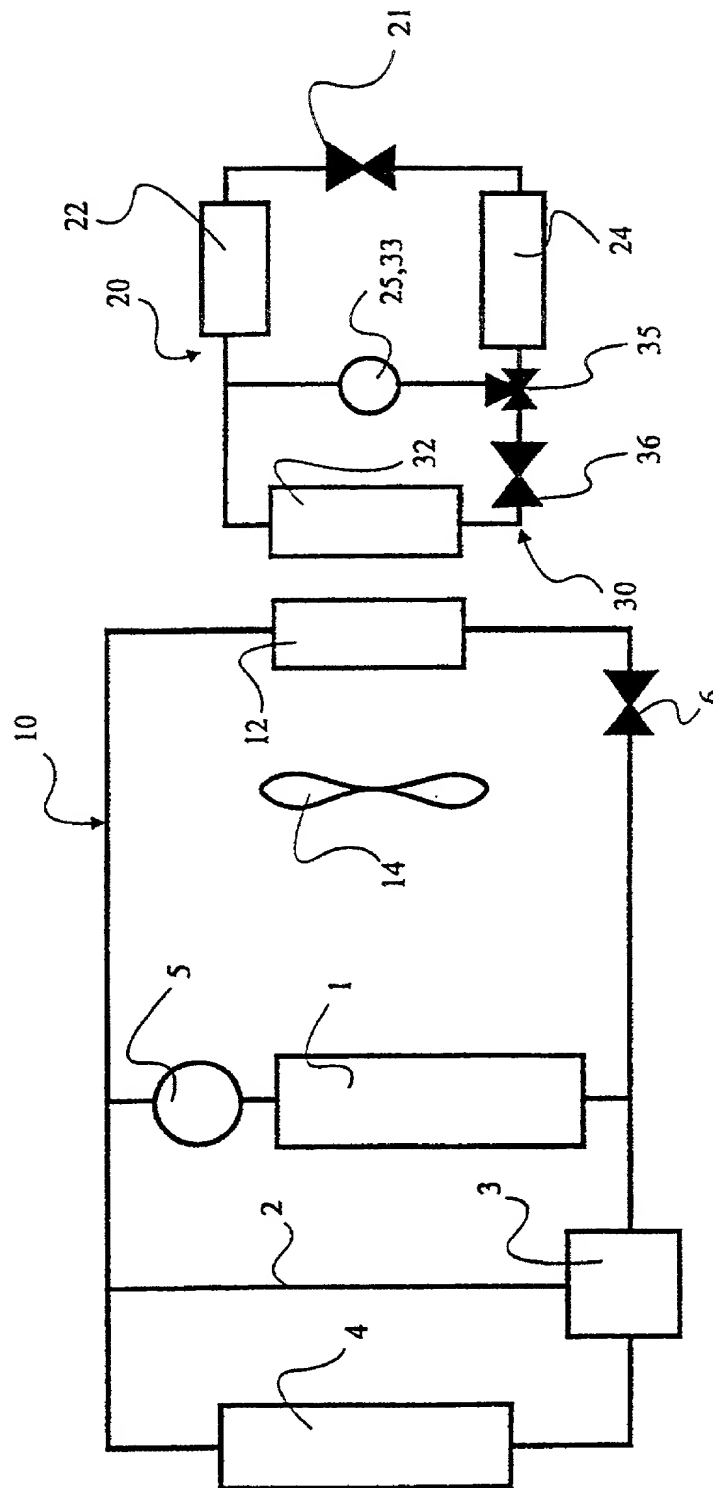


Fig. 2



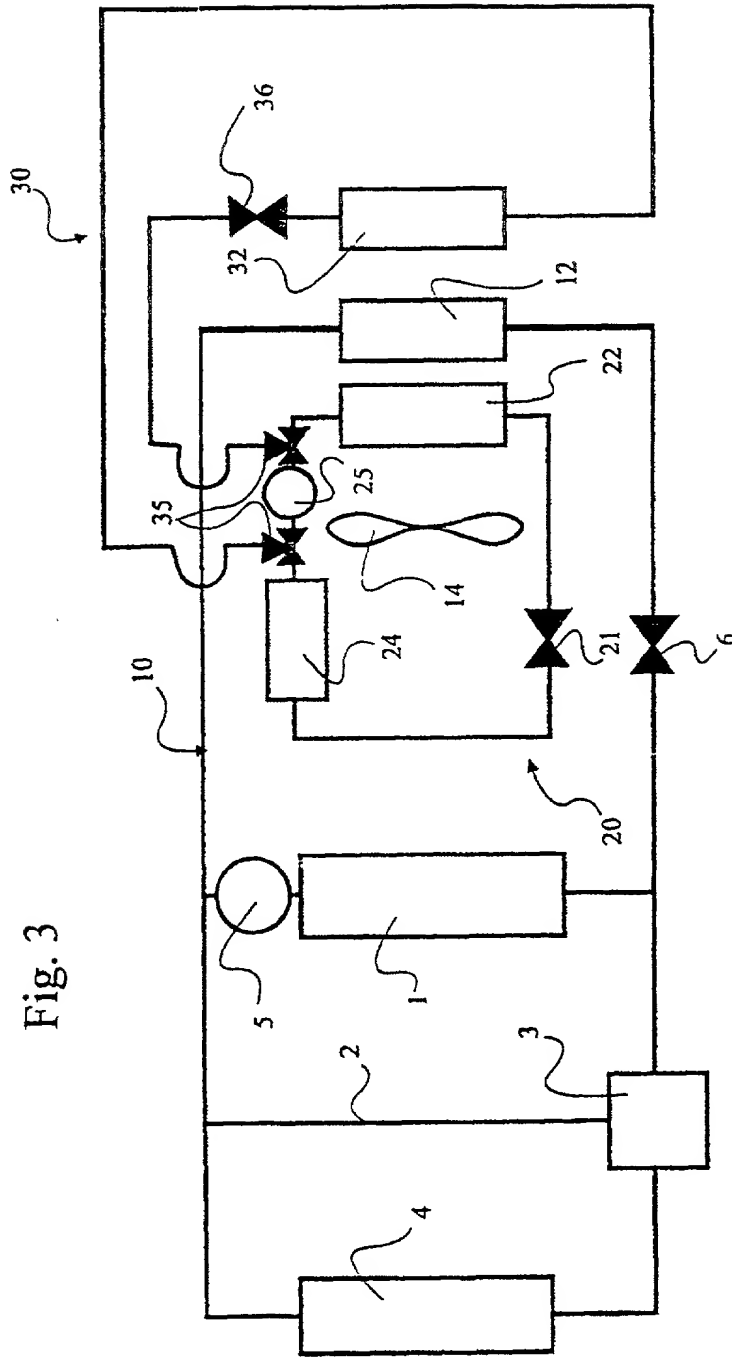


Fig. 3

Fig. 4

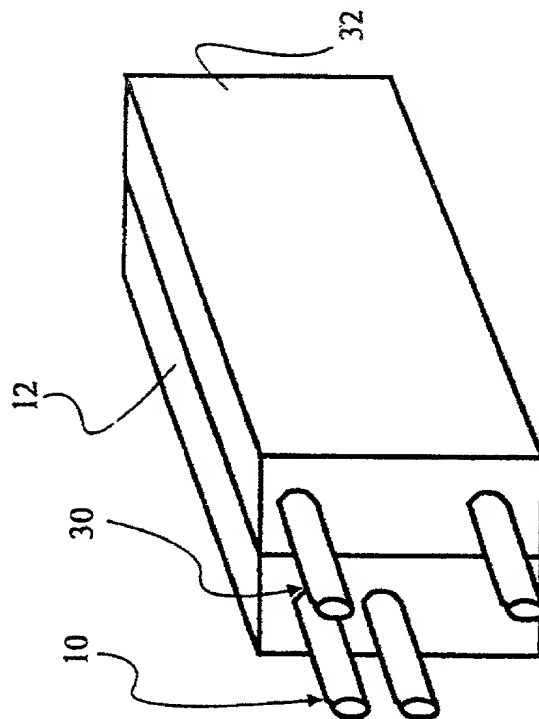




Fig. 5

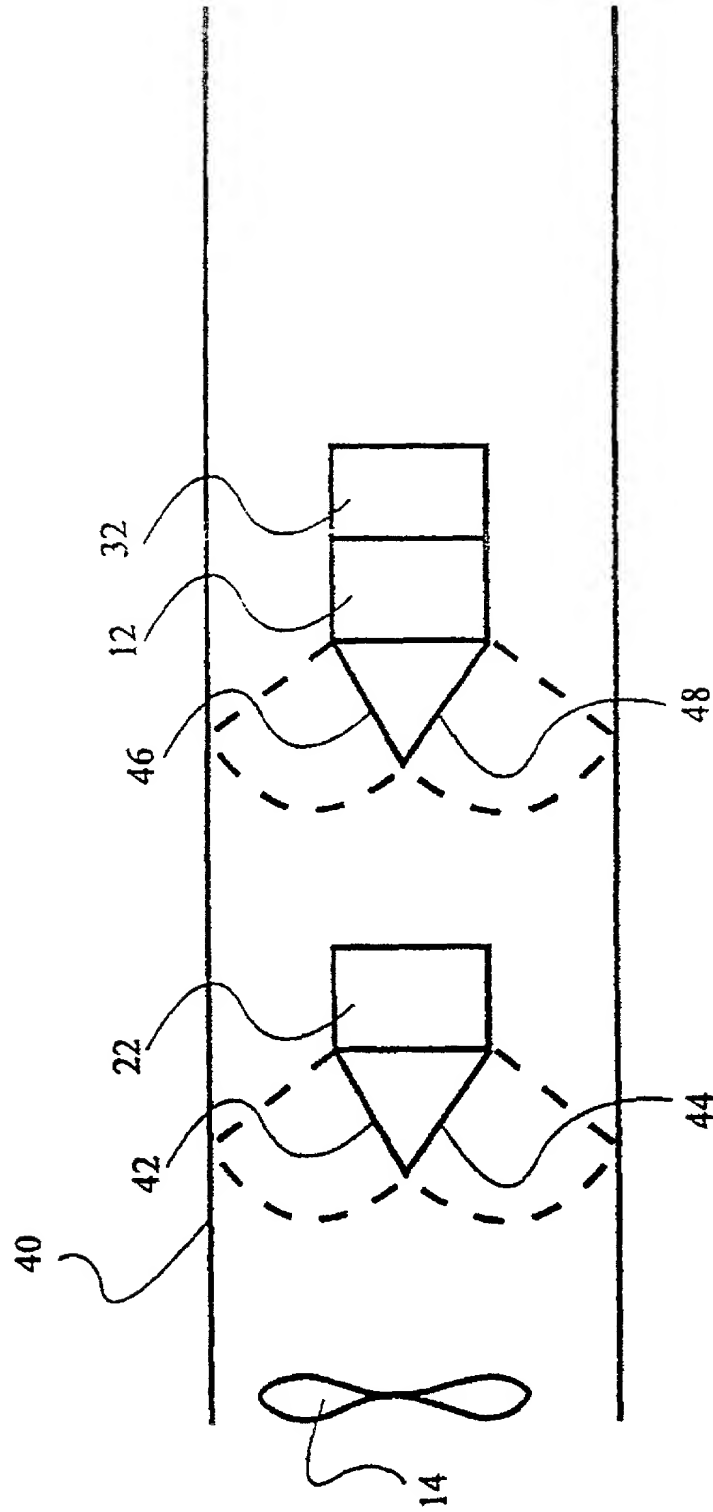


Fig. 6a

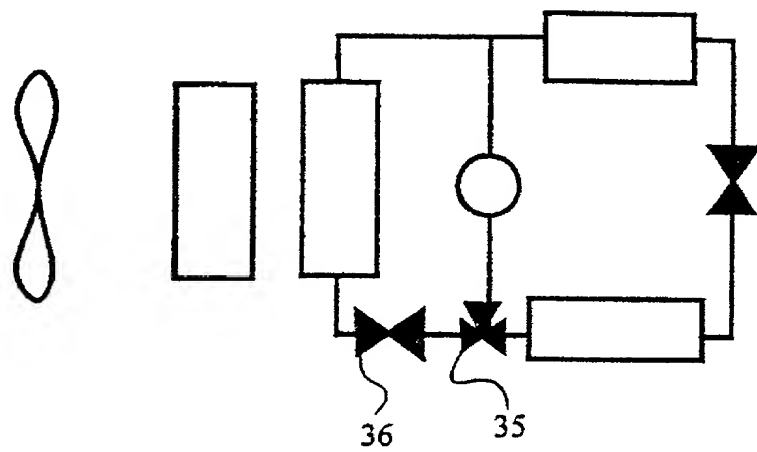


Fig. 6b

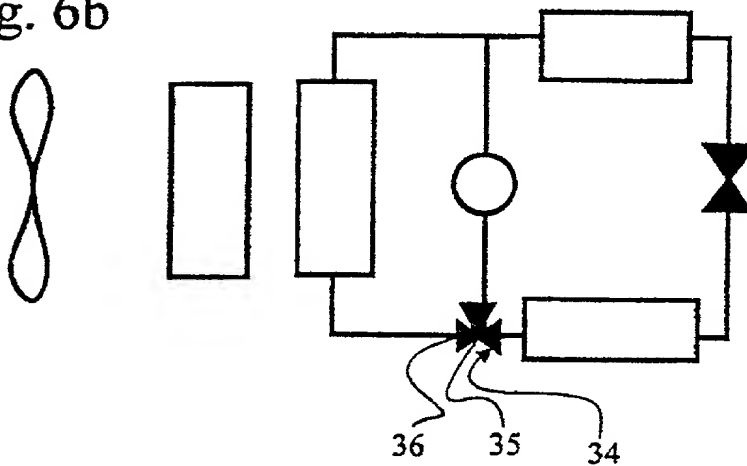


Fig. 6c

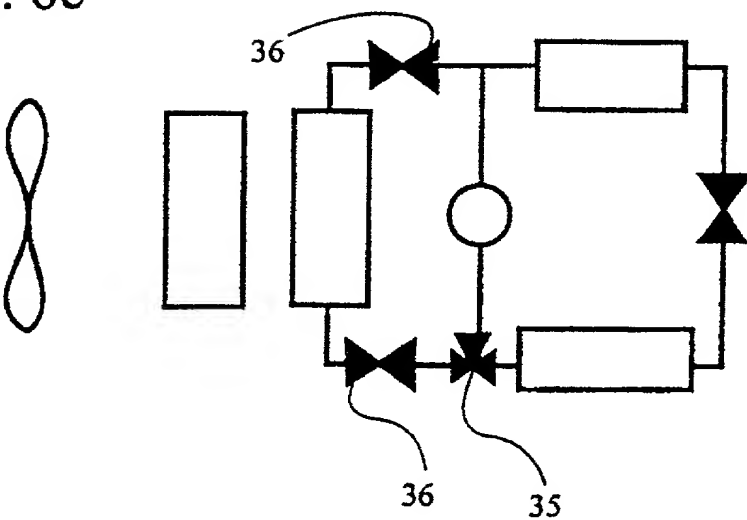


Fig. 6d

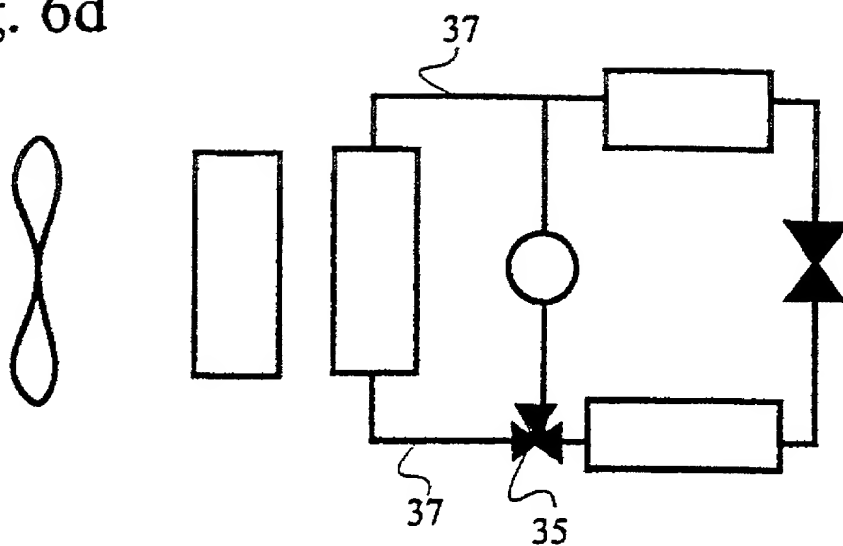
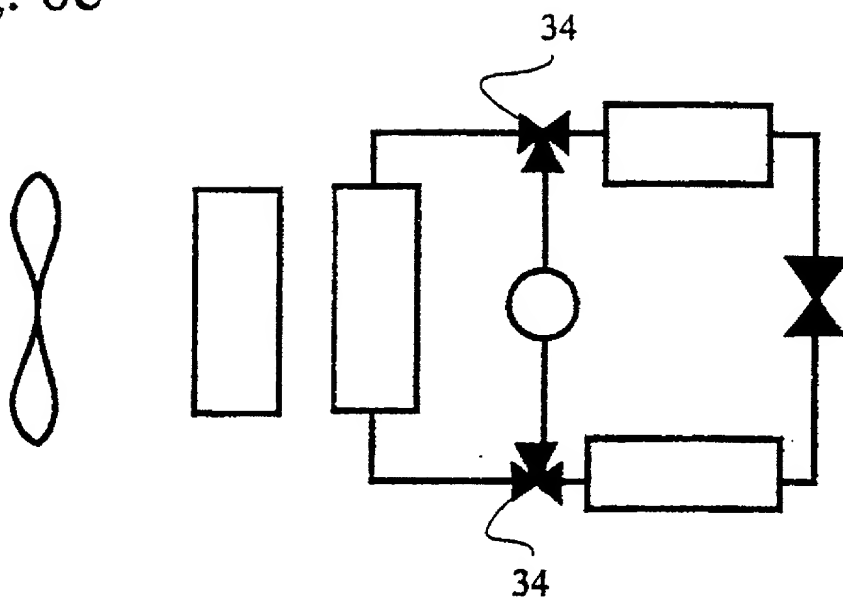


Fig. 6e



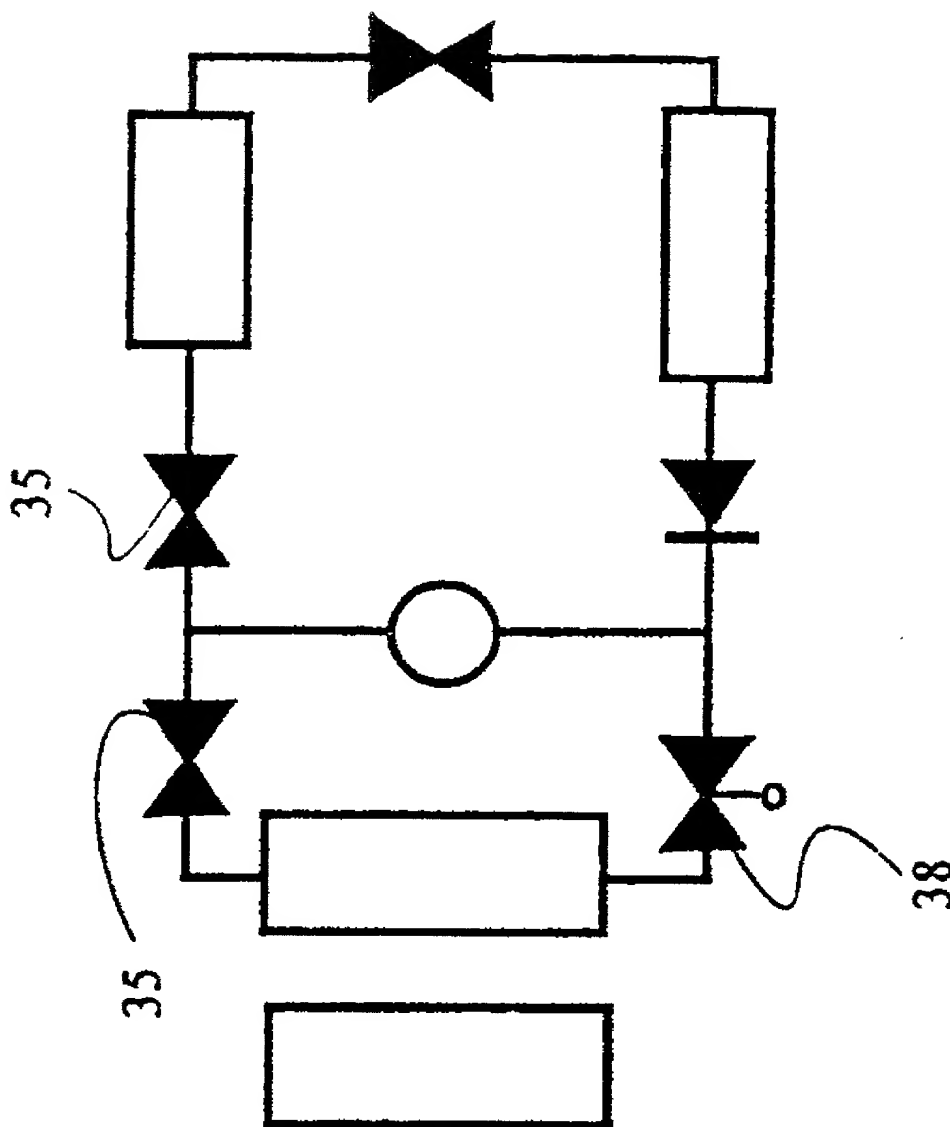


Fig. 7



As a below named inventor, I hereby declare that:

My residence, post office address and citizenship are as stated below next to my name.

27123

PATENT TRADEMARK OFFICE

I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled Heating system components, the specification of which

(check one) ☐ is attached hereto.

☐ was filed on \_\_\_\_\_ as  
Application Serial No. \_\_\_\_\_  
and was amended on \_\_\_\_\_ (if applicable).

I hereby state that I have reviewed and understand the contents of the above identified specification, including the claims, as amended by any amendment referred to above.

I acknowledge the duty to disclose information which is material to the examination of this application in accordance with Title 37, Code of Federal Regulations, §1.56(a).

I hereby claim foreign priority benefits under Title 35, United States Code, §119 of any foreign application(s) for patent or inventor's certificate listed below and have also identified below any foreign application for patent or inventor's certificate having a filing date before that of the application on which priority is claimed:

Prior Foreign Application(s)			Priority Claimed	
<u>199 21 812.9</u>	<u>GERMANY</u>	<u>11/05/1999</u>	Yes	No
(Number)	(Country)	(Day/Month/Year Filed)		
_____	_____	_____	Yes	No
(Number)	(Country)	(Day/Month/Year Filed)		
_____	_____	_____	Yes	No
(Number)	(Country)	(Day/Month/Year Filed)		

I hereby claim the benefit under Title 35, United States Code, §120 of any United States application(s) listed below and, insofar as the subject matter of each of the claims of this application is not disclosed in the prior United States application in the manner provided by the first paragraph of Title 35, United States Code, §112, I acknowledge the duty to disclose material information as defined in Title 37, Code of Federal Regulations, §1.56(a) which occurred between the filing date of the prior application and the national or PCT international filing date of this application:

<u>PCT/IB00/00722</u>	<u>May 5, 2000</u>	_____
(Application Serial No.)	(Filing Date)	(Status—patented, pending, abandoned)
_____	_____	_____
(Application Serial No.)	(Filing Date)	(Status—patented, pending, abandoned)

I hereby appoint the following attorney(s) and/or agent(s) to prosecute this application and to transact all business in the Patent and Trademark Office connected therewith:

Joseph A. Calvaruso (Reg. N° 28,287) (1)

SEND CORRESPONDENCE TO:

DIRECT TELEPHONE CALLS TO:  
(name and telephone number)

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I hereby declare that all statements made herein of my own knowledge are true and that all statements made on information and belief are believed to be true; and further that these statements were made with the knowledge that willful false statements and the like so made are punishable by fine or imprisonment, or both, under Section 1001 of Title 18 of the United States Code and that such willful false statements may jeopardize the validity of the application or any patent issued thereon.

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Full name of fourth joint inventor	_____	Date	_____
Fourth inventor's signature	_____	Citizenship	_____
Residence	_____		
Post Office Address	_____		